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EXACT SOLUTION OF THE THREE-DIMENSIONAL PROBLEM OF IDEAL PLASTICITY

S. I. Senashov

Let $r\theta_Z$ be a cylindrical coordinate system, σ_r , σ_θ , σ_Z , $\tau_{r\theta}$, τ_{rz} , τ_{θ_Z} be the components of the stress tensor, u, v, and w be the components of the velocity vector, and k be the yield stress under pure shear.

The equations of ideal plasticity with the von Mises yield condition are of the form

$$\begin{aligned} \frac{\partial \sigma_{r}}{\partial r} + \frac{1}{r} \frac{\partial \tau_{r\theta}}{\partial \theta} + \frac{\partial \tau_{rz}}{\partial z} + \frac{\sigma_{r} - \sigma_{\theta}}{r} = 0, \\ \frac{\partial \tau_{r\theta}}{\partial r} + \frac{1}{r} \frac{\partial \sigma_{\theta}}{\partial \theta} + \frac{\partial \tau_{\theta z}}{\partial z} + \frac{2\tau_{r\theta}}{r} = 0, \\ \frac{\partial \tau_{rz}}{\partial r} + \frac{1}{r} \frac{\partial \tau_{\theta z}}{\partial \theta} + \frac{\partial \sigma_{z}}{\partial z} + \frac{\tau_{rz}}{r} = 0, \\ (\sigma_{r} - \sigma_{\theta})^{2} + (\sigma_{\theta} - \sigma_{z})^{2} + (\sigma_{z} - \sigma_{r})^{2} + 6(\tau_{r\theta}^{2} + \tau_{rz}^{2} + \tau_{\theta z}^{2}) = 6k^{2}, \\ \lambda \frac{\partial u}{\partial r} = 2\sigma_{r} - \sigma_{\theta} - \sigma_{z}, \quad \lambda \left(\frac{u}{r} + \frac{1}{r} \frac{\partial v}{\partial \theta}\right) = 2\sigma_{\theta} - \sigma_{z} - \sigma_{r}, \\ \lambda \frac{\partial w}{\partial z} = 2\sigma_{z} - \sigma_{r} - \sigma_{\theta}, \quad \lambda \left(\frac{1}{r} \frac{\partial w}{\partial \theta} + \frac{\partial v}{\partial z}\right) = 2\tau_{\theta z}, \\ \lambda \left(\frac{\partial u}{\partial z} + \frac{\partial w}{\partial r}\right) = 2\tau_{rz}, \quad \lambda \left(r \frac{\partial}{\partial r} \left(\frac{v}{r}\right) + \frac{1}{r} \frac{\partial u}{\partial \theta}\right) = 2\tau_{r\theta}, \\ \frac{\partial}{\partial r} (ru) + \frac{\partial v}{\partial \theta} + r \frac{\partial w}{\partial z} = 0, \\ \sigma_{r} + \sigma_{\theta} + \sigma_{z} = 3p. \end{aligned}$$

We shall assume that

$$\mathbf{\tau}_{\mathbf{r}\mathbf{r}} = \mathbf{\tau}_{\mathbf{r}\mathbf{\theta}} = \mathbf{0}.\tag{2}$$

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We shall seek the solution of Eqs. (1) in the form

$$u = u^*(r) \text{sh} \, \xi, \, v = v^*(r) \text{ch} \, \xi, \, w = w^*(r) \text{ch} \, \xi, \, p = p(r), \, \xi = z + \beta \theta, \tag{3}$$

where u*, v*, w* are functions only of r and β is an arbitrary constant. Then we obtain from the incompressibility condition and Eqs. (2) a system of ordinary differential equations for determination of the functions u*, v*, w*:

$$u^{*} + \frac{dw^{*}}{dr} = 0, \quad r \frac{d}{dr} \left(\frac{v^{*}}{r} \right) + \frac{\beta}{r} u^{*} = 0, \quad \frac{d}{dr} (ru^{*}) + \beta v^{*} + rw^{*} = 0, \tag{4}$$

and the equation

$$d\sigma_r/dr + (\sigma_r - \sigma_{\theta})/r = 0$$
(5)

remains for the determination of the pressure p. The system of equations (4) reduces to the Bessel equation

$$r^{2}u^{*''} + ru^{*'} - (r^{2} + \beta^{2} + 1)u^{*} = 0.$$

The solution of this equation is of the form

$$u^* = C_1 I_v(r) + C_2 K_v(r), \ v = \sqrt{\beta^2 + 1}, \tag{6}$$

where I_v are the Bessel functions of imaginary argument, K_v is the MacDonald function, and C_1 and C_2 are arbitrary constants. If one sets $C_2 = 0$ in (6), the velocity field is of the form

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$$u = C_1 I_{\mathbf{v}}(r) \mathrm{sh} \,\xi, \qquad (7)$$

$$v = -\left[rC_1 \beta \int I_{\mathbf{v}}(r) \frac{dr}{r^2}\right] \mathrm{ch} \,\xi, \quad w = -C_1 \mathrm{ch} \,\xi \int I_{\mathbf{v}}(r) \,dr.$$

The components of the stress tensor are equal to

$$\sigma_{\mathbf{r}} = C + \int \frac{F(\varphi - 1)}{r} dr, \quad \tau_{\theta z} = \psi F,$$

$$\sigma_{\theta} = \sigma_{\mathbf{r}} + (\varphi - 1)F, \quad \sigma_{z} = \sigma_{\mathbf{r}} - (2 + \varphi)F, \quad F = \sqrt{2}k(1 + \varphi^{3} + (1 + \varphi)^{2} + (1/2)\psi^{2})^{-1/3},$$

$$\varphi = (u^{*} + \beta v^{*})/ru^{*'}, \quad \psi = (\beta w^{*} + rv^{*})/ru^{*'}.$$
(8)

One can use the solution (7) and (8) to describe the plastic flow of a cylinder (0 < r \leq R, $-l \leq z \leq l$), loaded at the ends by a stress distribution according to the law

$$\sigma_z = -(2+\varphi)F + \int_0^r \frac{F(\varphi-1)}{r} dr$$

and by the torsional moment

$$M = 2\pi \int_{0}^{R} \tau_{\theta z} r^2 dr.$$

Assuming the lateral surface to be free of stresses, we determine the constant C from the condition $\sigma_r(R) = 0$.

If $C_2 \neq 0$ in formula (6), one can use the solution constructed to describe the plastic flow of a tube acted on by tensile stresses, a torsional moment, and internal pressure.

If one sets $\beta = 0$ in formulas (8), the components of the stress tensor will coincide with those found in [1] for the case of axisymmetric strain.

One can also use the solution found to describe the flow of a plastically nonuniform medium; for this it is sufficient to set k = K(r) in formulas (1) and (8).

LITERATURE CITED

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